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25 moisture reduction in the product has valuable advantages such as reducing the microbial
26 activity and chemical deterioration besides the substantial reduction in the product volume
27 (**Doymaz & Pala, 2003**), the drying of food products consumes a considerable amount of
28 energy (**ElGamal et al., 2014; 2015**). Using renewable solar energy in drying process
29 reduces the use of conventional energy sources and as a result reduces the pollutant emissions
30 (**Santos et al., 2005**). Solar energy as the cleanest source of energy is also easily accessible
31 and abundantly available as an alternative energy source in nature (**Rao et al., 2017**). Using
32 solar drying for food preservation is one of the most optimistic applications of solar energy
33 (**Sekyere et al., 2016**). Indirect solar drying systems (solar dryers with a solar air heater)
34 provide better control of the characteristics of the drying air (**Sacilik et al., 2006; Sreekumar
35 et al., 2008**).

36 The performance of any solar system such as solar air heaters and solar panels depends
37 fundamentally on the intensity of the solar radiation and the incidence angle. Maximum
38 energy is produced by a solar system when it is positioned at right angle to the sun (**Yousef,
39 1999; Khan, 2010**). Moving the solar system to continuously face the sun radiation during
40 the whole day is an efficient approach to achieve the maximum efficiency of solar collection
41 compared to the fixed solar systems. To achieve high system efficiency, an automated system
42 (namely solar tracker) is required to continually adjust the system to the optimum position as
43 the sun traverses the sky. Commercial purposes of solar tracking systems are (1) increasing
44 solar system output, (2) improving the thermal efficiency, (3) maximizing the power per unit
45 area and (4) grabbing the energy throughout the day. The theoretical background and
46 intensive details for different solar trackers and their operating systems can be found in
47 **Banerjee (2015)**. Numerous experiments performed in employing solar tracking systems
48 revealed that up to 40% of additional energy can be attained (**Clifford and Eastwood, 2004**).
49 Studies showed also that tracking systems increased pumping capacity (**Bione et al., 2004**),

50 power collection efficiency (**Rizk and Chaiko, 2008**) and thermal efficiency (**Guihua et al.,**
51 **2012; Dhanabal et al., 2013**) over the fixed-angle system. Moreover, reports indicated that
52 the use of solar trackers can increase electricity production by about 15-35% (**Tiberiu et al.,**
53 **2012; Anyaka et al., 2013; Miloudi et al., 2013; Quesada et al., 2015; Watane and Dafde,**
54 **2013**) and up to 40% in some regions (**Mork and Weaver, 2009; Banerjee, 2015**) compared
55 with solar modules of a fixed orientation angle. For instance, **Anusha et al. (2013)** compared
56 the fixed and single axis solar tracking systems for six days and their results show that the
57 solar tracking system increased the efficiency of about 40% and the energy received was
58 improved from 9.00 to 18.00 h.

59 Despite such confirmed merits of using solar trackers for improving the performance of solar
60 systems (**Samimi-Akhijahani, & Arabhosseini, 2018; Devan et al., 2020**), incorporating
61 solar trackers with solar air heaters especially in drying agricultural products has received
62 little attention for practical applications. Thence, the main aim of this study was to develop an
63 integrated solar tracking system to maximize the conversion efficiency of the previously
64 designed recyclable aluminum cans (RAC) solar air heater by continually adjusting its
65 direction to the optimum angle towards the sun during daytime hours. The thermal efficiency
66 of the fixed and tracking solar collectors was explored at different air flow rates and the
67 drying behavior of peeled and unpeeled apple slices in both systems was investigated and
68 compared with the traditional ambient sun drying.

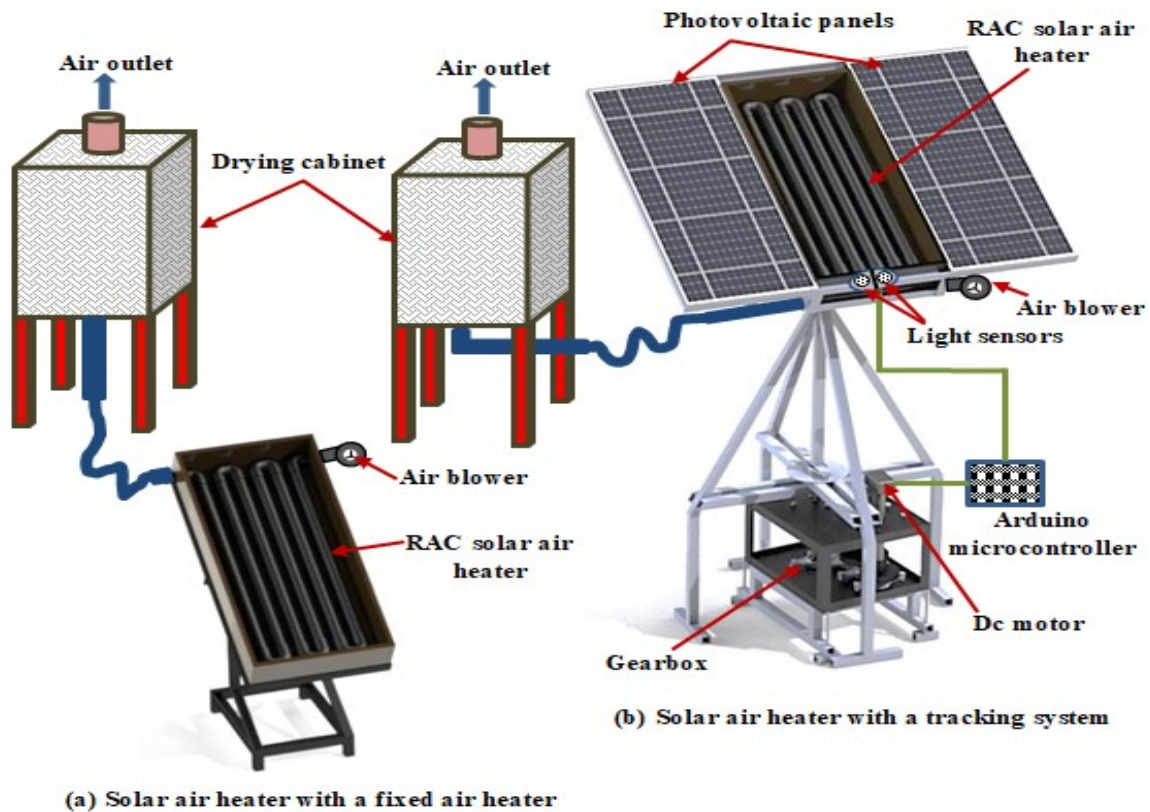
69 **2. MATERIALS AND METHODS**

70 **2.1. Solar dryer**

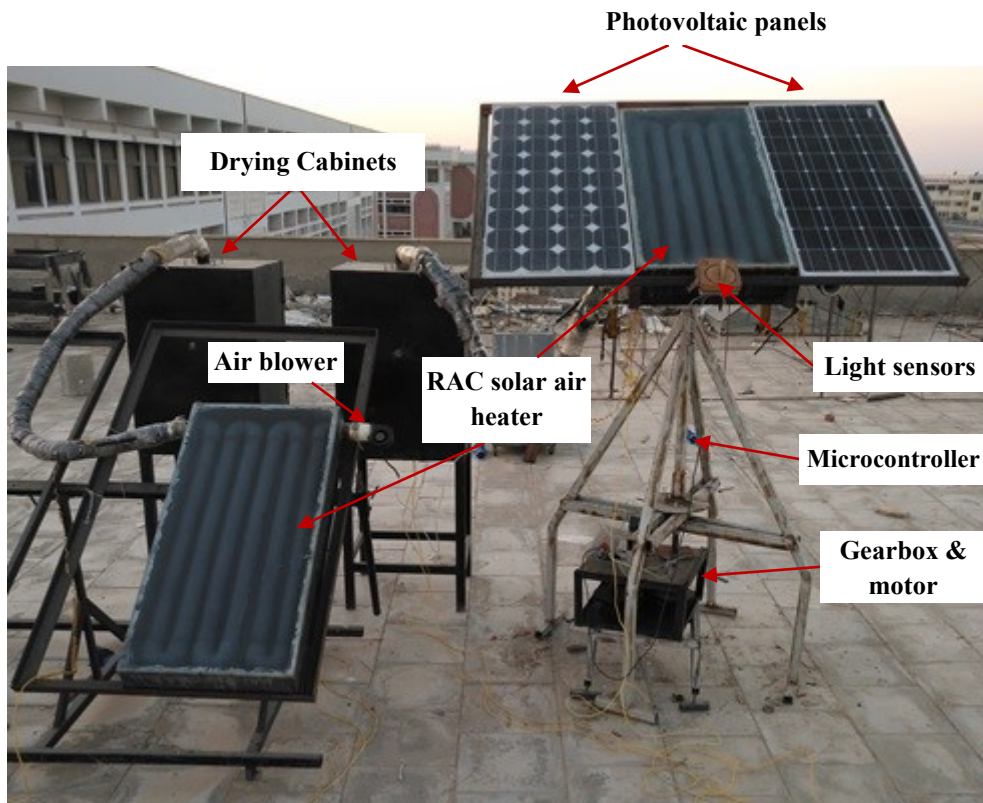
71 In our previous study (**Kishk et al., 2019**), an indirect solar dryer with forced convection was
72 designed and constructed at our lab in Suez Canal University, Egypt. In essence, the solar air
73 heater was fabricated from recycled aluminum cans (RAC) through which the ambient air

74 was heated and then forced inside drying cabinets. The RAC solar air heater was orientated in
 75 the East-West direction as the best orientation in Ismailia city, Egypt (latitude of 30.62°,
 76 longitude 32.27°). The solar heaters were placed facing south and inclined at an angle of 31°
 77 and kept in such a fixed position during the whole drying process.

78 In the current study, a solar tracking unit was developed to be incorporated with an RAC air
 79 heater in order to maximize its conversion efficiency by continually adjusting its orientation
 80 angle during the experiments. A drying cabinet to accommodate food samples to be dried was
 81 then attached to each air heater (The fixed and tracking ones). Two photovoltaic panels were
 82 also installed on the tracking system to provide the required power for operating air blowers
 83 and the motor of the tracking system itself as shown in Figure (1). The detailed information
 84 about the design of the RAC solar air heater was explained in details by **Kishk et al. (2019)**.



85
 86 Figure 1a. Schematic diagram for solar dryers integrated with solar air heaters at (a) fixed
 87 orientation (left side) and (b) attached with a solar tracking unit (right side).



88

89 Figure 1b. A photograph of solar air heaters integrated with drying cabinets

90 **2.2. Solar tracking system**

91 A single-axis tracking unit with one axis of rotation was particularly developed to be
 92 incorporated with the solar air heater. The two solar panels as well as the solar heater were
 93 fixed on the upper end of an iron shaft of the axis of rotation and the other end of the shaft
 94 was attached to the biggest gear in the gearbox to reduce the rotational speed incoming from
 95 the motor as the required movement step was very small in each cycle. The tracking module
 96 consists of light sensing devices, a control unit, and a driving mechanism as shown in Figure
 97 (1). Two light sensors were attached to the forefront frame of the solar collector (Figure 1) to
 98 measure the light intensity received from the sun. The solar collector of the air heater will
 99 remain in a certain position when light intensity received by both light sensors is equal. If
 100 light intensity received by one sensor was different from the other sensor (due to movement
 101 of the sun), it means the sun moved to a new position and the whole collector should be

102 rotated to face the sun in this new position. The signals from light sensors were transferred to
103 a microcontroller unit to operate the DC motor of a driving mechanism for moving the solar
104 air heater to this new position. The driving mechanism supported with a differential gearbox
105 and driven by the DC motor was responsible for moving the whole unit to a new position
106 until both sensors receive equal light intensity again.

107 **2.3. Experimental setup**

108 Drying of apple slices using a fixed and a tracking solar air heater system was conducted
109 during July, 2019 under clear climate conditions. Three experimental runs were conducted for
110 drying apple slices (var. Red delicious) under three air flow rates of 22, 33 and 44 m³h⁻¹.
111 Before conducting each experiment, fresh apple fruits were washed by running water and
112 some fruits were peeled before being sliced. The peeled and unpeeled apples were cut
113 horizontally into slices with approximately 6.0±1 mm in thickness. To prevent browning of
114 apples slices prior to solar drying, all raw slices were immersed in ascorbic and citric acid
115 solution (10 g ascorbic acid + 2 g citric acid dissolved in one liter of distilled water)
116 (Pizzocaro et al., 1993). Apple slices were then kept on a mesh grid for 5 min at the room
117 temperature (25 °C) to drain all excessive immersion solution before being dried in the
118 dryers. The slices were arranged over 50 × 50 cm trays in four rows (2 rows of peeled slices
119 and 2 rows of unpeeled slices) with a net weight of about 300 g for each tray. Three trays full
120 of apple slices were prepared for each experimental run: two trays to be dried inside the fixed
121 and tracking solar dryers besides one tray was used for traditional sun drying (control).
122 Before each experimental run, the desired air flow rate was adjusted using a digital blower
123 speed controller (SMB-10, USA). The experimental work was run for 10 hours continuously
124 through the period from 8 am to 6 pm, solar time. During the experimental work, an electrical
125 digital balance (BS-Series, China) with an accuracy of 0.001 g was used to determine the
126 mass of wet and dry samples to calculate moisture content. The temperature and the relative

127 humidity of the drying air were continuously recorded during the experiments using a data
 128 logger (Lab-Jack logger, USA) connected with a computer supported with instantaneous data
 129 acquisition software (Weather link, USA). The output data were recorded every five minutes
 130 and averaged every one hour. Climate conditions such as air speed, solar radiation and dew-
 131 point temperature were recorded by a meteorological station (Vantage Pro 2, Davis, USA)
 132 installed in the experimental location. Moreover, a pyranometer was used for measuring the
 133 solar irradiance at the collectors and expressed as flux density (W/m²). The technical
 134 specifications for all devices used in this study are presented in Table 1.

Table 1. Technical specifications of the used instrumentations and measurement devices.

Device or instrument	Quantity	Technical specifications
Photovoltaic panels	2	Maximum power = 300W each Power voltage = 12V
Tracking motor	1	DC motor, 12V, 50W
Air blower	2	DC fan, 12 V, 40W
Light sensors	2	Lab designed
Anemometer	1	SMB-10, USA, measuring range up to 30 ms ⁻¹ and accuracy of ± 0.1 ms ⁻¹
Thermocouples	8	K-type -200:1200 °C ±0.5 °C
Data logger	1	Lab-Jack logger, USA
Digital balance	1	BS-Series, China, with an accuracy of 0.001 g

135 2.4. Thermal efficiency determination

136 To examine the effectiveness of using the solar tracking system for improving the
 137 performance of the RAC solar systems, thermal efficiency was determined under the tested
 138 air flow rates. Thermal efficiency of the solar heating systems (η) was calculated using the
 139 following equation (Kishk et al., 2019):

$$140 \quad \eta = \frac{\dot{m} \cdot C_p(T_0 - T_i)}{I \cdot A_c} \quad (1)$$

141 where: \dot{m} is the air mass flow rate (kg s^{-1}), C_p is the specific heat of the air ($\text{J Kg}^{-1} \text{K}^{-1}$), T_o is
142 the outlet air temperature (K), T_i is the inlet air temperature (K), I is the total solar radiation
143 incident upon the plate of the solar collector (Wm^{-2}), and A_c is the area of collector (m^2).

144 **2.5. Drying process and data analysis**

145 To obtain the drying curves of apple slices, samples of peeled and unpeeled apple slices were
146 weighed before drying and constantly monitored at different times during drying experiments
147 to record weight loss. The initial moisture content of apple samples determined using the
148 oven-drying method (Zlatanović et al., 2013) was 86.12% wet basis (6.2 kg water/kg dry
149 matter). Based on the initial moisture content and weight loss data recorded during drying
150 process in both systems, the moisture content of apple slices at various stages during drying
151 operation can be easily estimated. Due to the fluctuation of the drying air temperature during
152 solar drying process, the average drying rate can be determined from the following formula at
153 every tested flow rate of the drying air using the measured values of moisture contents:

$$154 \quad DR = \frac{M_0 - M_f}{\Delta t} \quad (2)$$

155 where DR is the average drying rate ($\text{kg H}_2\text{O/kg dry solids/hour}$), M_0 and M_f are the initial
156 and final moisture contents ($\text{kg water/kg dry matter}$), respectively, Δt is the drying period (10
157 h).

158 **2.6. Estimation of effective moisture diffusivity (D_{eff})**

159 As declared in Eq. 3, Fick's second law of diffusion can be used to explain the drying process
160 of apple since moisture diffusion across the spatial dimension (Z) is one of the main mass
161 transfer phenomena that describes drying process of agro-food materials (Doymaz, 2007).

162
$$\frac{\partial MR}{\partial t} = D_{eff} \frac{\partial^2 MR}{\partial Z^2} \quad (3)$$

163 The moisture ratio (MR) that relates the gradient of the sample moisture content at a certain
 164 time (M_t) to both the initial (M_0) and the equilibrium moisture contents (M_e) can be
 165 calculated using Eq. 4.

166
$$MR = \frac{M_t - M_e}{M_0 - M_e} \quad (4)$$

167 where D_{eff} is the effective moisture diffusivity ($m^2 s^{-1}$), M_t is the moisture content at a time t
 168 of drying (kg water/kg dry matter), M_e is the equilibrium moisture content (kg water/kg dry
 169 matter). The value of the equilibrium moisture content is relatively small compared to M_t or
 170 M_0 . Thus equation 4 can be safely simplified to $MR=M_t/M_0$ (**Doymaz & Pala, 2003; Kishk**
 171 **et al., 2019**).

172 Assuming that temperature and diffusion coefficients are constant during drying, moisture
 173 migration is only caused by diffusion and shrinking is negligible, thus, the solution of Fick's
 174 law for a thin plate (apple slice) can be given as follow (**Baroni & Hubinger, 1998**):

175
$$MR = \frac{8}{\pi^2} \sum_{n=0}^{\infty} \frac{1}{(2n+1)^2} \exp \left[-\frac{(2n+1)^2 \pi^2 D_{eff} t}{4L^2} \right] \quad (5)$$

176 where L is the half thickness of slab (m) and n is the number of terms in the diffusion cycles.

177 As the thickness of the apple slice was quite small (0.006 m) and the drying time was
 178 relatively large, Eq. (5) can be further simplified to only the first term ($n = 0$) of the series

179 (**Tutuncu & Labuza, 1996**). Hence, Eq. (5) is rewritten in a logarithmic form as follows:

180
$$\ln MR = \ln \frac{8}{\pi^2} - \frac{\pi^2 D_{eff} t}{4L^2} \quad (6)$$

181 Therefore, the slope of the linear relationship between the dependent variable ($f(t) =$
182 $\ln MR$) and the drying time (t) was then used to estimate the effective moisture diffusivity
183 (D_{eff}) according to Eq. (7):

$$184 \quad \text{Slope} = -\frac{\pi^2 D_{eff}}{4L^2} \quad (7)$$

185 2.7. Drying models

186 Mathematical modeling is an important step in proper design of the dryers to better
187 understanding the mechanism of drying processes. Page model (Velić et al., 2004; Kaleta et
188 al., 2013) as well as Henderson & Pabis model Zlatanović et al. (2013) were used to
189 describe the drying kinetics and the changes in moisture content of apple slices during drying
190 process:

$$191 \quad \text{Page model:} \quad MR = \exp(-kt^n) \quad (8)$$

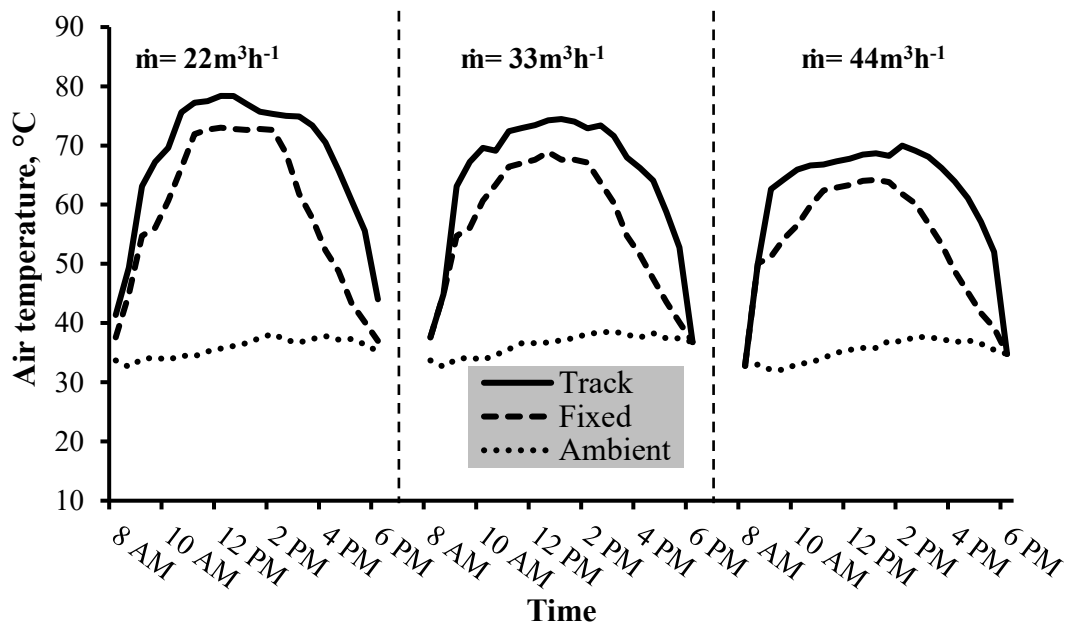
$$192 \quad \text{Henderson & Pabis model:} \quad MR = a \exp(-kt) \quad (9)$$

193 The parameters k , n and a were calculated by using least squares regression solved by a
194 Quasi-Newton numerical method. The correlation coefficient (R^2), reduced chi-square (χ^2)
195 and root mean square error ($RMSE$) were used as measures of model robustness (Kishk et al.,
196 2019).

197 3. RESULTS AND DISCUSSION

198 During drying period, the ambient air temperature ranged from 32 to 38 °C with relative
199 humidity values ranged from 40 to 60 %. The solar radiation values varied during the drying
200 period where the minimum values recorded was observed at the beginning and at the end of
201 the drying time (8 am and 6 pm) with a minimum value of 220 Wm⁻². Meanwhile, the
202 maximum values of the solar radiation were recorded at noon (11-1 pm) with a minimum

203 value of 870 Wm^{-2} . Figure 2 illustrates the variation of air temperature of the ambient and
 204 drying air in the fixed and tracking systems at different air flow rates of 22, 33, 44 m^3h^{-1} . It is
 205 quite clear that the solar air heater equipped with a tracking unit heats up the air to
 206 temperatures higher than that in the fixed solar air heater at all tested air flow rates. This
 207 implies that the tracking system maximized the benefits from the solar radiation incident on
 208 the solar collector resulting in higher air temperatures during the whole drying period.



209
 210 Figure 2. Variation of air temperature during the drying time at different air flow rates (22,
 211 33, 44 m^3h^{-1}).

212
 213 It can be also seen from Figure 2 that the outlet air temperature decreased as the air flow rate
 214 increased. For example, the maximum temperature of the outlet air from the fixed solar heater
 215 was 72.8, 68.9, 64.2 $^{\circ}\text{C}$ at the air flow rate of 22, 33 and 44 m^3h^{-1} , respectively. Similarly, the
 216 maximum temperature of the outlet air from the tracking solar heater reached 78.5, 74.4,
 217 70.0 $^{\circ}\text{C}$ at the air flow rate of 22, 33 and 44 m^3h^{-1} , respectively (Figure 2). This might be
 218 ascribed to the longer residence time for the air inside the solar collector at lower air flow
 219 rates (Alta et al., 2010; Kishk et al., 2019).

220 **3.1. Thermal efficiency**

221 The thermal efficiency of the fixed and tracking solar heaters was calculated according to
 222 equation (1) based on the temperature and weather data recorded during the drying
 223 experiments at different air flow rates and the results are presented in Table 2. It is clear that
 224 the thermal efficiency of the solar air heater was enhanced significantly to about 45% when
 225 the tracking system was used. At all tested air flow rates, the solar air heater with a tracking
 226 system provided higher thermal efficiency than the fixed solar air heater. For example, at the
 227 air flow rate of $22 \text{ m}^3\text{h}^{-1}$, the average thermal efficiency realized by the fixed solar air heater
 228 was 35%. Meanwhile, the average thermal efficiency increased up to 50.9% with heaters
 229 equipped with a tracking system at the same air flow rate (Table 2).

230 The results also showed that as the airflow rate increased the thermal efficiency of the solar
 231 air heater substantially improved (Table 2). For example, when the airflow rate increased
 232 from 22 to $44 \text{ m}^3\text{h}^{-1}$, the average thermal efficiency of the fixed solar air heater augmented
 233 from 35 to 56.1%, respectively. Meanwhile, the corresponding values of the average thermal
 234 efficiency of the solar air heater with a tracking system increased from 50.9 to 80.7% when
 235 air flow rate increased from 22 to $44 \text{ m}^3\text{h}^{-1}$, respectively. This trend of the results is in
 236 agreement with those of the previous studies reported for the RAC solar air heaters (**Ozgen et**
 237 **al., 2009; Kishk et al., 2019**).

Table 2. Thermal efficiency (%) of the fixed and tracking solar air heaters operated at different air flow rates.

System	$\dot{m} = 22 \text{ m}^3\text{h}^{-1}$		$\dot{m} = 33 \text{ m}^3\text{h}^{-1}$		$\dot{m} = 44 \text{ m}^3\text{h}^{-1}$	
	Fixed	Tracking	Fixed	Tracking	Fixed	Tracking
Min	15.6	44.8	24.4	60.8	27.3	71.0
Max	42.6	61.0	54.1	75.4	64.2	87.1
Average	35.0	50.9	47.3	68.3	56.1	80.7

238 3.2. Drying process of apple slices

239 Apple slices were dried for 10 hours (from 8am to 6pm) inside solar dryers operated under
240 fixed and tracking solar air heaters as well as in the ambient air (control). The experiments
241 were repeated under different air flow rates of 22, 33, 44 m³h⁻¹. The resulted drying curves of
242 apple slices were then illustrated as shown in Figure 3. As a general trend, the drying rate of
243 apple slices (either peeled or unpeeled) in the tracking unit was considerably higher than
244 those in either fixed system or the ambient sun drying. For instance, after 10 hours (*i.e.* 600
245 min) of drying at the airflow rate of 22 m³h⁻¹, the moisture content of the peeled apple slices
246 decreased from 86.12% w.b. to 7.7 % w.b. for the tracking system; while, at the same air flow
247 rate the final moisture content after the same period was still at 16.9 and 18% w.b. for the
248 fixed system and the ambient drying, respectively. The high drying rate of apple slices
249 realized in the tracking system could be ascribed to the higher temperature values of the
250 drying air passed from the solar heater to the drying cabinet compared to those in the fixed
251 ambient systems as illustrated in Figure 2. In essence, when drying temperature increased the
252 effective moisture diffusion increased resulting in higher drying rate as explained later in
253 section 3.4. Although apple slices were dried faster in the tracking system and eventually
254 reached constant moisture content earlier, the system were kept working for the whole drying
255 hours (*i.e.* 10 h) to facilitate the comparison with the other two drying systems operated at the
256 same drying period (Figure 3).

257 On the other hand, Figure 3 shows that the drying curves of apple slices in the fixed solar
258 heater and in the open sun drying (control) were close to each other at the lower air flow rate
259 (*i.e.* 22 m³h⁻¹). The drying rate of apple slices in the fixed system was higher than that of the
260 ambient unit by increasing airflow rate as shown in Figure 3. These findings agree with those
261 reported in our previous study on tomato drying (Kishk et al., 2019). This implies that the
262 fixed solar air heater is not preferred to be used in drying applications at low air flow rates

263 without improving the heater's thermal efficiency by employing different scenarios such as
 264 incorporating a tracking system. In all cases, the drying curves of apple slices (either peeled
 265 or unpeeled) within the tracking system were much better than that of the fixed and the
 266 control dryers under all values of air flow rates. This implies that dryers equipped with a
 267 tracking system have higher thermal efficiency and consequently will have efficient drying
 268 capacity of apple slices.

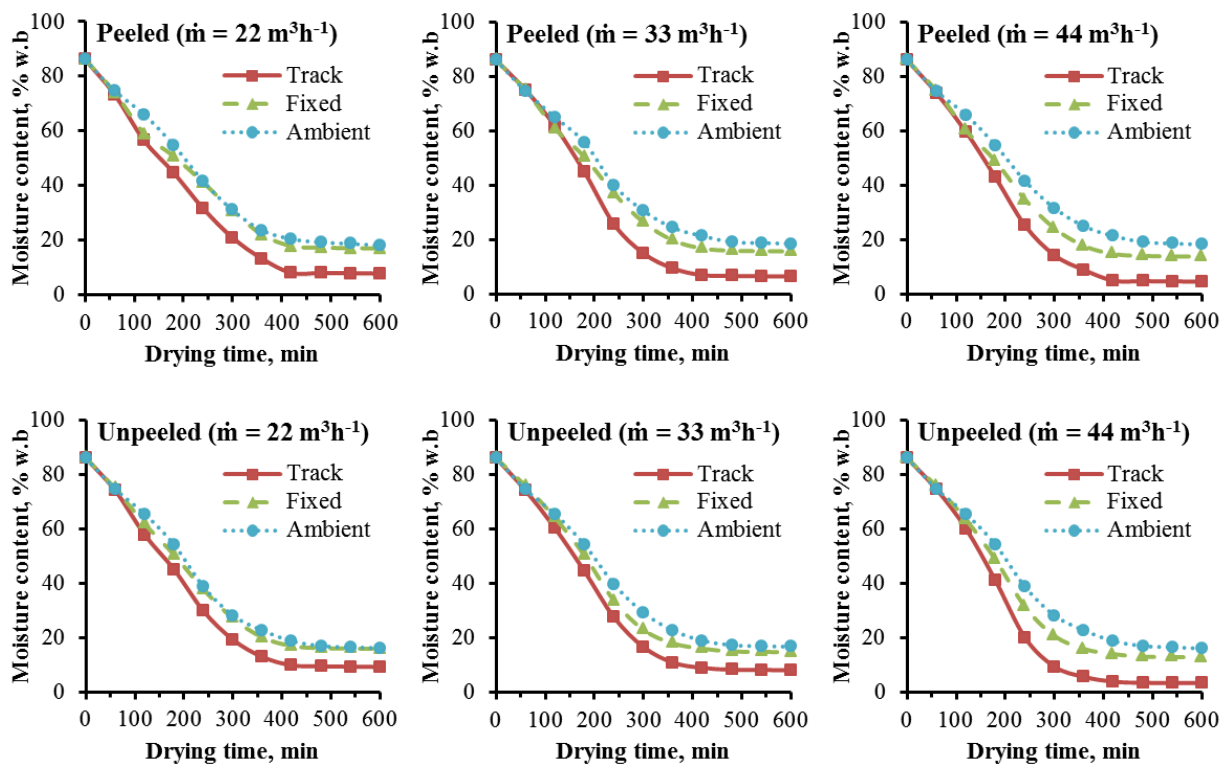


Figure 3. Drying curves of peeled and unpeeled apple slices dried in three drying systems (tracking, fixed and ambient) at different air flow rates.

269 To demonstrate the influence of air flow rate on the drying behavior of apple slices, Figure 4
 270 presents the drying curves of unpeeled apple slices at different air flow rates for the fixed and
 271 tracking systems (same trend showed for peeled slices). For comparing drying behavior of
 272 apple slices using different drying systems at different air flow rates, the average drying rate
 273 DR ($\text{kg H}_2\text{O/kg dry solids/hour}$) defined as the difference between the initial and final
 274 moisture contents ($\text{kg H}_2\text{O/kg dry solids}$) divided by the drying time (10 h) was calculated

275 and the results are tabulated in Table 3. In general, when the air flow rate increased, the
 276 drying rate also increased in both tested systems. Although the air temperatures delivered
 277 from the solar heaters were lower at higher flow rates (Figure 2), the dry air mass at high
 278 flow rates helped in removing the evaporated moisture from the apple slices quickly resulting
 279 in a higher drying rate. The increase of air flow rate from 22 to 33 m³h⁻¹ slightly increased the
 280 drying rate from 0.610 to 0.611 (kg H₂O/kg dry solids/hour) for tracking system and from
 281 0.601 to 0.603 (kg H₂O/kg dry solids/hour) for fixed system as shown in Table 3. Meanwhile,
 282 further increase in the air flow rate from 33 to 44 m³h⁻¹ resulted in a considerable increase in
 283 the drying rate within the tracking system from 0.611 to 0.617 H₂O/kg dry solids/hour. As a
 284 consequence, the moisture content of apple slices at the end of drying period (10 working
 285 hours) in the tracking system was much lower compared to that in the fixed system for all
 286 tested flow rates as explicitly depicted in Figure 4. For instance, moisture content of apple
 287 slices dried in the tracking system at a flow rate of 44 m³h⁻¹ reached 3.3 % at the end of the
 288 drying period compared to only 12.8% in the fixed system operated at the same flow rate.
 289 The same trend was also observed in the other tested flow rates as shown in Figure 4.

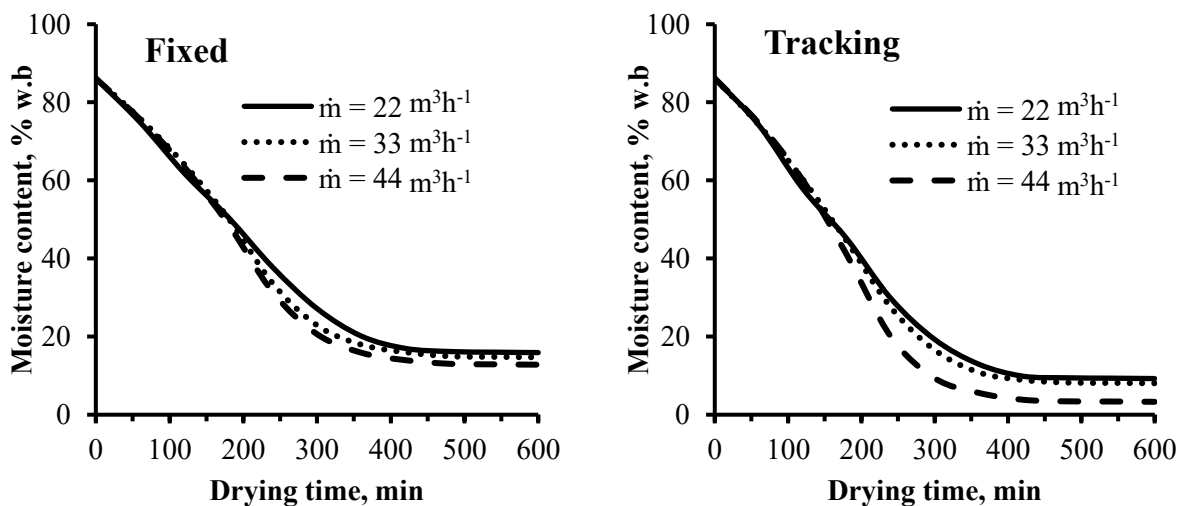


Figure 4. Effect of air flow rate on drying behavior of unpeeled apple slices for fixed and tracking drying systems

Table 3. Average drying rate DR (kg H₂O/kg dry solids/hour) of apple slices

Setup	System	Unpeeled slices		Peeled slices	
		Tracking	Fixed	Tracking	Fixed
1	$\dot{m} = 22 \text{ m}^3\text{h}^{-1}$	0.6103	0.6015	0.6121	0.6001
	Ambient*	0.6011		0.5995	
2	$\dot{m} = 33 \text{ m}^3\text{h}^{-1}$	0.6117	0.6032	0.6135	0.6018
	Ambient*	0.6004		0.5987	
3	$\dot{m} = 44 \text{ m}^3\text{h}^{-1}$	0.6170	0.6058	0.6156	0.6044
	Ambient*	0.6014		0.6001	

*The ambient drying was conducted under the ambient air conditions of temperature and air speed.

290 Analysis of variance (ANOVA) was conducted for each individual system first to see the
 291 difference in drying rate between peeled and unpeeled apple slices. Another ANOVA test
 292 was performed to see the difference in drying rate among the three drying systems (tracking,
 293 fixed and ambient) at different flow rates (22, 33 and 44 m³h⁻¹). The analysis of variance
 294 (ANOVA) tests showed that there was no significant difference ($p \geq 0.05$) in drying rate for
 295 peeled and unpeeled apple slices dried in three drying systems at all tested conditions. These
 296 results are in agreement with **Defraeye and Radu (2018)** who reported that the peel is a
 297 barrier for moisture transport as a result of the reduced surface area for evaporation, but it is
 298 not a barrier for heat transport. However, they found that the differences between the peeled
 299 and unpeeled apple slices during drying were quite small due to the fact that the moisture
 300 below the peel can also escape partially via the side surface. This result was due to the small
 301 thickness of the slices used in their study (5mm) similar to the current study (6mm). For
 302 thicker apple slices, the impact of the peel is expected to become more pronounced.
 303 Paradoxically, there was a significant difference ($p \leq 0.05$) in drying rate among the three
 304 modes of drying as well as among the different air flow rates (Table 3). Least significant
 305 difference (LSD) test was used to compare drying rate of the three examined systems. From
 306 Table 3, one can also conclude that at a certain air flow rate, the drying rate of apple slices in

307 the tracking system was significantly different ($p \leq 0.05$) than that in the fixed one.
308 However, drying in the fixed system at low air flow rate $22 \text{ m}^3\text{h}^{-1}$ showed no significant
309 difference with the ambient air drying. For a given drying system, the drying rate at air flow
310 rate of $44 \text{ m}^3\text{h}^{-1}$ was significantly higher than the drying rate at the other tested air flow rates
311 (22 and $33 \text{ m}^3\text{h}^{-1}$) and this was reflected in the moisture content curves shown in Figure 3 and
312 Figure 4.

313 3.4. Moisture diffusivity

314 To analyze mass transfer phenomenon during drying process, it was important to investigate
315 the condition of the effective moisture diffusivity (D_{eff} , $\text{m}^2 \text{ s}^{-1}$) because this feature affects the
316 rate of water vapor transfer from inside the slice to its surface and because it represents most
317 of the parameters influencing the drying rate (equation 5). The values of effective moisture
318 diffusivity of apple slices during drying are tabulated in Table 4. The values of D_{eff} reported
319 in this study as shown in Table 4 are in the range of those values reported for different
320 cultivars of apple and different drying scenarios. The reported D_{eff} values lie between 1.7×10^{-10}
321 $- 4.4 \times 10^{-10} \text{ m}^2\text{s}^{-1}$ for apple *cv. Jonagold* (Velić et al., 2004), $0.48 \times 10^{-10} - 2.02 \times 10^{-10} \text{ m}^2\text{s}^{-1}$
322 for apple *cv. Red Delicious* (Kaya, 2007) and $0.32 \times 10^{-10} - 1.53 \times 10^{-10} \text{ m}^2\text{s}^{-1}$ for apple *var.*
323 *Granny Smith* (Vega-Gálvez et al., 2012). As a general trend in both fixed and tracking
324 drying systems, the moisture diffusivity of peeled and unpeeled apple slices increased with
325 increasing the air flow rate as shown in Table 4. The values of moisture diffusivity of apple
326 slices dried in solar dryer equipped with a tracking system were significantly higher than
327 those in the fixed solar dryer and ambient drying. The highest value of D_{eff} ($5.43 \times 10^{-10} \text{ m}^2 \text{ s}^{-1}$)
328 in apple slices was obtained in the solar dryer equipped with a tracking system at the highest
329 air flow rate ($44 \text{ m}^3\text{h}^{-1}$), while the lowest value of D_{eff} ($2.42 \times 10^{-10} \text{ m}^2\text{s}^{-1}$) was recorded for the
330 peeled slices dried in the ambient air.

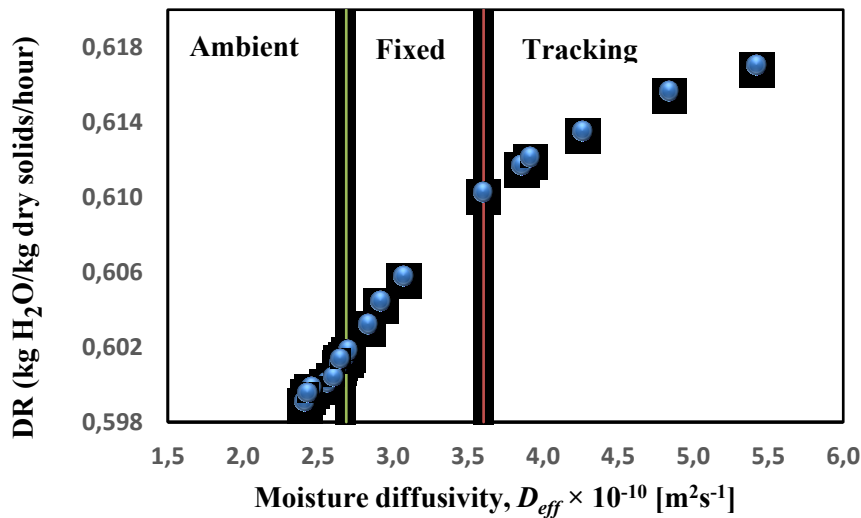
Table 4. Effective moisture diffusivity $D_{\text{eff}} \times 10^{-10} [\text{m}^2\text{s}^{-1}]$ of apple slices during drying in different drying systems.

Setup	System	Unpeeled slices		Peeled slices	
		Tracking	Fixed	Tracking	Fixed
1	$\dot{m} = 22 \text{ m}^3\text{h}^{-1}$	3.60	2.69	3.92	2.56
	Ambient*		2.65		2.46
2	$\dot{m} = 33 \text{ m}^3\text{h}^{-1}$	3.86	2.84	4.27	2.71
	Ambient*		2.61		2.42
3	$\dot{m} = 44 \text{ m}^3\text{h}^{-1}$	5.43	3.08	4.84	2.92
	Ambient*		2.65		2.43

*The ambient drying was conducted under the ambient air conditions of temperature and air speed.

331 As the effective moisture diffusivity is sensitive to any change in the drying parameters
332 (**Zlatanović et al., 2013**), the variation in the drying air temperatures among the examined
333 drying systems (Figure 2) resulted in different values of moisture diffusivity of apple slices in
334 these systems (**Diamante, 1994**). Figure 5 shows the relationship between the moisture
335 diffusivity and the drying rate of apple slices in different drying systems. It is clear to observe
336 that drying rate increases with the increase of moisture diffusivity in all tested drying
337 systems. Apple slices in the tracking system had the highest values of moisture diffusivity,
338 and as a result, the highest values of drying rate as shown in Figure 5. The highest values of
339 moisture diffusivity realized in the tracking system could be ascribed to the higher
340 temperature values of the drying air passed from the solar heater to the drying cabinet
341 compared to those in the fixed system and the ambient sun drying. In essence, when the
342 drying temperature increased, the moisture diffusion of agricultural products increased
343 resulting in higher drying rate (**Doymaz, 2007; Brooks et al., 2008; Vega-Gálvez et al.,**
344 **2012; Kishk et al., 2019**). It is also important here to highlight the effectiveness of the
345 tracking system for providing the highest values of D_{eff} and drying rate. Most importantly, the

346 high drying rate during drying process of agricultural products reduces the energy
 347 requirements and the time required for accomplishing drying processes.



348
 349 Figure 5. Moisture diffusivity vs. drying rate of apple slices in different drying systems.

350 3.4. Drying models

351 Drying data of apple slices expressed as the moisture ratio (*MR*) versus drying time were
 352 fitted to **Page** and **Henderson & Pabis** models. As there was no significant difference found
 353 between the drying rate of peeled and unpeeled apples slices as explained earlier, drying data
 354 were only modeled for unpeeled slices. The drying model constants and the statistical
 355 parameters namely the coefficient of determination (R^2), the root mean square error (*RMSE*)
 356 and the reduced chi-square (χ^2) used to evaluate the goodness of fit for solar drying of apple
 357 slices are listed in Table 5. It is clear from the calculated statistical parameters that both
 358 selected models provide a good description of the experimental data under different systems
 359 and air flow rates particularly for tracking-based system. It can be also indicated that the Page
 360 model gave the higher value of R^2 (0.994) and the lower values of *RMSE* (0.027) and χ^2
 361 (0.0009) for the drying data of the tracking system at the high air flow rate of 44 m³h⁻¹.
 362 Meanwhile, the Henderson & Pabis model gave the highest value of R^2 (0.985) and the lower
 363 values of *RMSE* (0.032) and χ^2 (0.0013) for the drying data of the tracking system at the low
 364 air flow rate of 22 m³h⁻¹ as shown in Table 5. Based on these results, Both selected models

365 (Page model; Henderson & Pabis model) can be satisfactory describe the drying behavior
 366 of apple slice under different drying conditions. These results are in agreement with findings
 367 reported by Velić et al. (2004) and Kaleta et al. (2013) who used Page model successfully to
 368 describe the drying kinetics of apple. Also, Zlatanović et al. (2013) who selected the
 369 Henderson & Pabis model as the suitable model to represent the drying characteristics of
 370 apple.

371 **Table 5.** Statistical results obtained from selected thin-layer drying models of apple slices

Model	\dot{m} (m ³ h ⁻¹)	System	Model constants	R ²	RMSE	χ^2	
Page model $MR = \exp(-kt^n)$	22	Tracking	k=0.171; n=1.277	0.983	0.035	0.0015	
		Fixed	k=0.1689; n=1.1163	0.973	0.039	0.0019	
		Ambient	k=0.1477; n=1.1616	0.972	0.041	0.0021	
	33	Tracking	k=0.1525; n=1.3932	0.982	0.038	0.0018	
		Fixed	k=0.1603; n=1.1873	0.961	0.051	0.0032	
		Ambient	k=0.1476; n=1.1533	0.973	0.040	0.0020	
	44	Tracking	k=0.1089; n=1.8159	0.994	0.027	0.0009	
		Fixed	k=0.1587; n=1.2414	0.964	0.050	0.0032	
		Ambient	k=0.1474; n=1.1631	0.973	0.040	0.0020	
	Minimum				0.961	0.027	0.0009
	Maximum				0.994	0.051	0.0032
	Henderson & Pabis model $MR = a \exp(-kt)$	22	Tracking	a=1.1971; k=0.3054	0.985	0.032	0.0013
Fixed			a=1.0994; k=0.226	0.978	0.036	0.0016	
Ambient			a=1.1118; k=0.2165	0.976	0.038	0.0018	
33		Tracking	a=1.2512; k=0.3294	0.979	0.039	0.0019	
		Fixed	a=1.1565; k=0.2508	0.969	0.045	0.0025	
		Ambient	a=1.1043; k=0.2124	0.976	0.038	0.0018	
44		Tracking	a=1.4059; k=0.4175	0.971	0.051	0.0032	
		Fixed	a=1.1882; k=0.2734	0.971	0.045	0.0025	
		Ambient	a=1.1123; k=0.2167	0.976	0.038	0.0018	
Minimum				0.969	0.032	0.0013	
Maximum				0.985	0.051	0.0032	

372 4. CONCLUSIONS

373 A solar tracking system was fabricated to be incorporated with a solar air heater
374 manufactured from recyclable aluminum cans (RAC) to enhance its thermal efficiency and to
375 examine its performance in drying apple slices. The tracked solar system enhanced the
376 efficiency of the solar air heater to achieve a maximum efficiency of 87.1% instead of 64.2%
377 for the fixed solar air heater at the same air flow rate. The tracking system increased the
378 moisture diffusivity of apple slices to reach a highest value of $5.43 \times 10^{-10} \text{ m}^2 \text{ s}^{-1}$. Also, drying
379 rate of apple slices in the tracking system was significantly higher compared with both the
380 fixed system and ambient drying at all tested air flow rates. The drying rate (*DR*) increased
381 with the drying air temperature and flow rate and the highest value of *DR* (0.617 H₂O/kg dry
382 solids/hour) was obtained in the dryer equipped with a tracking module at highest air flow
383 rate of 44 m³h⁻¹. The results revealed that at the lowest air flow rate of 22 m³h⁻¹, the drying
384 rate of apple slices was equal for fixed system and ambient air drying. Thus, it can be
385 conclude that the solar air heater is not preferred to be used in drying applications at low air
386 flow rates without enhancing its thermal efficiency using different methods such as tracking
387 system or air circulation. Tracking systems are very effective in providing high values of both
388 moisture diffusivity and drying rate. The high drying rate during drying process of
389 agricultural products reduces the energy requirements and the time required for
390 accomplishing drying processes.

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